

# Reinforcement Effects Obtained by Applying Composite Material Sleeves to Repair Transmission Pipelines

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*This paper presents the results of the research work performed by the authors aiming at highlighting and quantifying the reinforcement effects (restoring of the mechanical strength) on transmission pipelines, generated by applying composite material sleeves in the areas in which the steel pipes of these lines have local surface defects (metal loss – like flaws). The models proposed to that purpose have been confirmed by performing some experimental programs, the first one consisting of the examination of the behaviour of some full-scale steel pipe specimens, with metal loss – like flaws (made by machining), on which reinforcing wraps made of composite material have been applied (with polymeric matrix and fiberglass fabric), subjected to internal pressure loading, up to bursting, and the second one aiming at the determination of the state of stress and strain in the pipelines on which reinforcing wraps made of composite material have been applied and which are subjected to internal pressure loading, in the elastic range. The items discussed and the results presented in the paper are mainly useful for the development and qualification of the composite materials repair systems for the transmission pipelines, according to the requirements of the standards ASME-PCC2 and ISO 24817.*

**Keywords:** Transmission pipeline, Local metal loss – like defects, Repair by applying composite material sleeves, Reinforcement effects of composite sleeves / wraps

The transmission pipelines intended for hydrocarbons (named in the followings HTP), whose steel pipes have local surface defects of the type metal loss, produced by means of internal and/or external corrosion processes (Locally Thin Areas – LTA, Pittings – PIT, Grooves – GLF or Gauges – GAU), can be repaired, with or without removing the pipelines from service, using a wide variety of technological procedures [1, 2].

Because HTPs must usually ensure the continuous supply (without interruptions or, eventually, with scheduled short-time interruptions) for the clients of the transported fluids (petroleum, petroleum products, natural gas etc.), the repair technologies developed for in-service (under pressure) pipelines are preferred. The in-service repair technologies which require welding operations are not preferred, because they use concentrated heat sources, and the possible burn through of the HTP wall while executing the repair works can have serious consequences, the transported fluids (petroleum, liquid petroleum products, natural gas etc.) having an increased flammability and exploding potential. Therefore, the in-service repair technologies consisting of the application of composite materials wraps / sleeves are perceived as being advantageous alternative solutions for substituting the classical technologies, based on the repair by welding of the areas with defects or on the application by welding of steel patches, wraps or sleeves in the damaged HTP areas [1-3].

Previous research works [1, 2] have shown that the quality level of the repairs performed by applying, in the damaged HTP areas, reinforcing wraps / sleeves of different types (metallic sleeves of type A or type B, metallic wraps filled with polymeric materials or composite materials wraps) is in direct correlation with the measure in which they guarantee: a) the restoration of the HTP mechanical strength; b) the taking over

of the HTP mechanical loadings (reduction of the stresses generated in the damaged pipe wall / with defects); c) the inhibition of the growth processes of the defects dimensions and diminution of their stress concentrator effects.

The analysis of the behaviour of a HTP upon which a composite wrap – CW has been applied, having a polymeric matrix (polyester, vinyl ester, epoxy or polyurethane) reinforced with fibres (glass, carbon, aramid, polyester or similar material), have led to the drawing of the schemes from figure 1, build considering that the steel from which HTP pipes are manufactured has a true/effective stress – strain characteristic curve,  $\sigma = f(\epsilon)$ , of the type “with linear strain hardening”. The investigation of these schemes highlights the following issues regarding the technical performances of the repairs with CW: a) for the HTP normal operating conditions (at a pressure  $p_0 \leq \text{MAWP}$ , MAWP – Maximum Allowable Working Pressure of HTP), which assumes its loading in the elastic region, the degree of taking over the mechanical loads by the CW and of unloading of the HTP damaged areas is in direct dependance with the ratio  $\rho_E = E_{CW} / E_{SP}$ ,  $E_{CW}$  being the elasticity modulus of the composite material from which the CW is made ( $E_{CW} = \text{tg}\alpha_{CW}$ ), and  $E_{SP}$  – elasticity modulus of the steel from which HTP pipes are manufactured ( $E_{CW} = \text{tg}\alpha_{SP}$ ); using the information summarised in table 1 regarding the mechanical properties of the composite materials which are presently used with priority to make CWs [4, 5], it results that  $\rho_E = 0.05 \dots 0.20$  (0.350) and, as a consequence, the CW takes over on a small scale the HTP mechanical loads; b) in the conditions of HTP overloading, which presupposes the occurrence of plastic deformations in its damaged areas, CW is gradually loaded, and the degree of taking over the mechanical loads by the CW could become equal or even greater than the one corresponding to the HTP; if the overloading is large enough,

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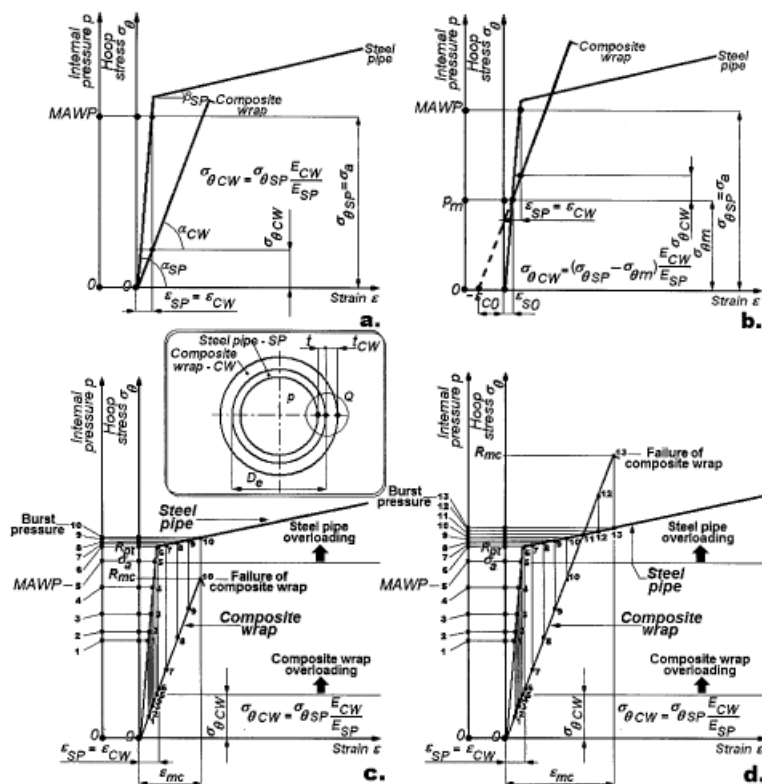


Fig. 1. Schemes for the analysis of the technical performances of the repairs with CW

Property <sup>e)</sup>	Composite material of CW <sup>a)</sup>						
	Perma Wrap <sup>b)</sup>	Fiba Roll <sup>b)</sup>	Clock Spring <sup>b)</sup>	RKIT 4D	RES-Q Wrap <sup>b)</sup>	IWR	KPB
Tensile modulus $E_{CW}$ , GPa	34.0... 38.0	7.9... 8.7	33.8... 34.5	48.0... 49.3	67.5... 69.8	17.5... 22.7	2.8... 3.1
Poisson's ratio $\mu_{CW}$ , -	0.30... 0.32 <sup>c)</sup>	0.15... 0.23	0.22... 0.25	0.18... 0.19	0.30... 0.33	0.40... 0.47	0.27... 0.30
Shear modulus $G_{CW}$ , GPa	3.1... 6.5 <sup>c)</sup>	-	3.1... 5.9	4.2... 5.5	6.5... 6.8 <sup>c)</sup>	1.8... 2.3 <sup>c)</sup>	-
Tensile strength $R_{mCW}$ , Mpa	580... 620	72... 190	630... 650	188... 205	822... 1020	265... 315	30... 31
Elongation at break $\varepsilon_{mCW}$ , %	1.0... 1.1	2.8... 3.7	1.0... 1.2	1.3... 1.4	0.25 <sup>d)</sup>	1.8... 2.2	1.1... 1.3
Adhesion to steel $A_{adCW}$ , Mpa	-	min. 11	-	-	-	12.5... 14.9	-

**Table 1**  
THE MAIN MECHANICAL  
PROPERTIES OF THE COMPOSITE  
MATERIALS FOR CW

a) Perma Wrap, Fiba Roll, Click Spring, IWR and KPB are armed with fibreglass, RKIT 4D – with aramid fibres, and RES-Q Wrap – with carbon fibres; b) these types of materials are delivered in different alternatives / grades (for instance, Fiba Roll delivers the CW grades: VECR, VECR HS1, VEFR, ISO etc.), the table presenting the ranges which frame the properties values existing in the catalogues and presentation documents for all grades belonging to the same type of composite material; c) these values have been determined based on the similarity with the values of the same properties for similar composite materials; the values are informative, as the degree of anisotropy of the materials is not precisely known; d) the value represents the allowable circumferential strain for CW; e) the values of the mechanical properties are for the CW circumferential direction.

failure will occur in the CW and then in the HTP damaged areas, because the CW elongation at break  $\varepsilon_{mc}$  is inferior to the one corresponding to the steel from which HTP pipes are manufactured  $\varepsilon_{ms}$  ( $\varepsilon_{mc} \ll \varepsilon_{ms}$ ); c) if CW is applied upon a HTP subjected to a pressure  $p_m$  (fig. 1.b), and then the operating pressure is increased at a value  $p_o$  ( $p_m < p_o \leq MAWP$ ), the HTP behaviour and the CW reinforcement effects will be as mentioned before; however, if the HTP will be operated at  $p_o < p_m$ , the mechanical co-operation between CW and HTP will depend upon the preservation of the CW adherence on the external surface of the HTP pipes.

## Experimental part

In order to underline the measure in which the issues previously presented (obtained on the ground of examining the schemes from fig. 1), regarding the technical performances of the repairs using CW, are confirmed

experimentally, several specimens of the type full-scale segment of steel pipe – TP have been constructed, upon which CWs have been applied in the areas where local defects of the type metal loss have been executed by machining. The general layout of TP, which has been provided with  $n_d = 1...3$  areas with defects reinforced by applying some CWs, is presented in figure 2. Table 2 and Figure 3 present the configuration and dimensions of the defects made by machining and the dimensions of CW applied on each specimen. The method of applying CWs on the specimen is shown in Figure 4, with the following specifications: a) CWs applied on TP1...TP3 have been of the IWR type, and CWs applied on the specimens TP4 and TP5 have been of KPB type (with the mechanical properties included in table 2); to cover the defects, before applying the CW, it has been used a polymeric filler – PF (reactive polymeric system modified with flexibility additive and mineral filler): ICECHIM type, for making





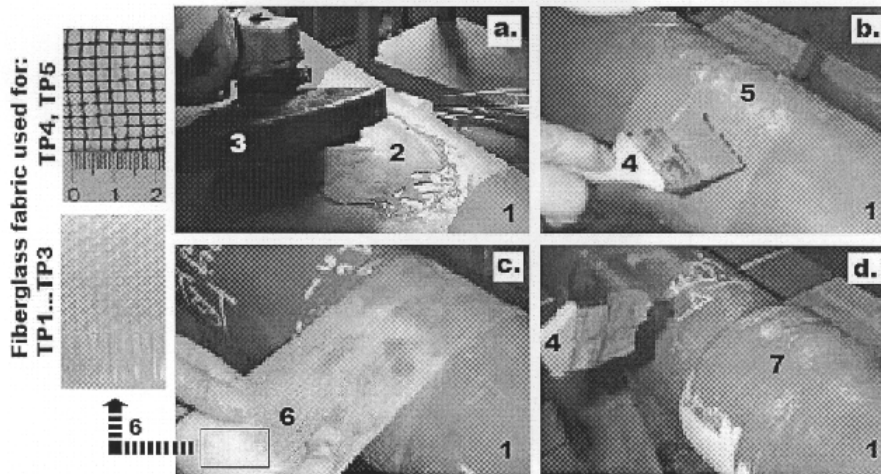


Fig. 4. The method of applying a CW in the TP areas with defects: a. PF application on the defect and smoothing of the SP surface; b. application of the first layer of the polymeric matrix PM of the CW; c. d. obtaining the CW by applying successive layers made of polymeric matrix PM reinforced with fibreglass fabric FW  
1. steel pipe of TP; 2. polymeric filler applied on the defect; 3. manual grinder; 4. brush; 5. polymeric matrix; 6. fibreglass fabric; 7. repair wrap made of composite material CW

where TP4 and TP5 have been built and tested), obtaining on such basis the information regarding: a) the value of the burst pressure  $p_b$ ; b) the position and mode of failure occurrence for each specimen; c) the peculiarities of the behaviour of the CW applied upon each specimen and the characterization of their reinforcement effect in the SP areas with defects.

In order to study experimentally the reinforcement effects of a CW applied upon a steel pipe subjected to internal pressure loading (in the elastic range), a specimen of the type full-scale segment / section of steel pipe (without local surface defects of the type metal loss) – TPT has been made using a segment of a L290 / X42 steel pipe (having the mechanical properties, determined experimentally by performing tensile tests on samples of the type pipe strip cut longitudinally from the SP used to build the TPT,  $R_{0.2} = 327$  MPa and  $R_m = 445$  MPa) – SP with two ellipsoidal ends – EE welded at its extremities, onto which a composite wrap – CW of IWR type has been applied (with the procedure used to build TP1...TP5 – fig. 4). On the external surface of the steel pipe – SP, with the external diameter  $D_e = 168.3$  mm, the length  $L_s = 620$  mm and the wall thickness  $t = 7.1$  mm, used to make the TPT, the strain gauges – TERs  $LG_{sp}$  (in the longitudinal direction) and  $CG_{sp}$  (in the circumferential direction) have been applied, and upon

the composite wrap – CW, with the length  $l_{cw} = 200$  mm and the thickness  $t_{cw} = 5.8$  mm, positioned on the TPT at the quota  $A_{cw} = 225$  mm, the TERs  $LG_{cw}$  (in the longitudinal direction) and  $CG_{cw}$  (in the circumferential direction) have been applied, as it can be seen in figure 6. All used TERs have been of the type 1-LY11-10/120, manufactured by Hottinger Baldwin Messtechnik – HBM.

TPT (filled with water) has been subjected to the internal pressure testing, aiming at, by recording the signals received from the TERs  $LG_{sp}$ ,  $CG_{sp}$ ,  $LG_{cw}$  and  $CG_{cw}$ , obtaining information regarding the amplitude of the mechanical stresses and strains generated in the components SP and CW. The internal pressure, recorded with the help of the pressure gauge / digital manometer – DM (type P3MBP – 1000, manufactured by HBM), has been progressively increased up to the level  $p_{max} = 24$  MPa, defined in such way that the components SP and CW of TPTs will be loaded only in the elastic range. During testing, the signals received from the DM and TERs have been acquired (with the speed of 5 values per second) and processed with the help of a system made of a Quantum X – SM device (manufactured by HBM), provided with an amplification module MX840 (universal amplifier achieving the power supply of the gauges, their signal conditioning, data acquisition and transmission towards the PC, ensuring the linearity for the range 0...12500 mm/m of TER deformations) and a computer

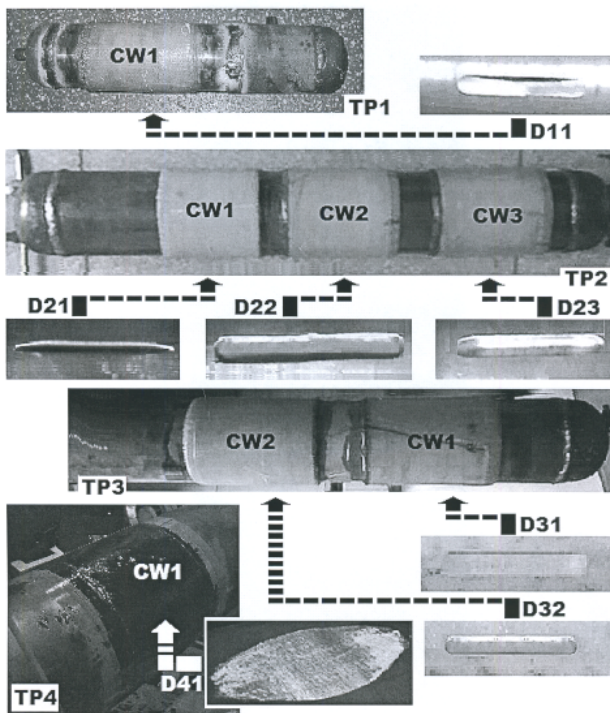


Fig. 5. Layout of TP prepared for the internal pressure testing up to bursting

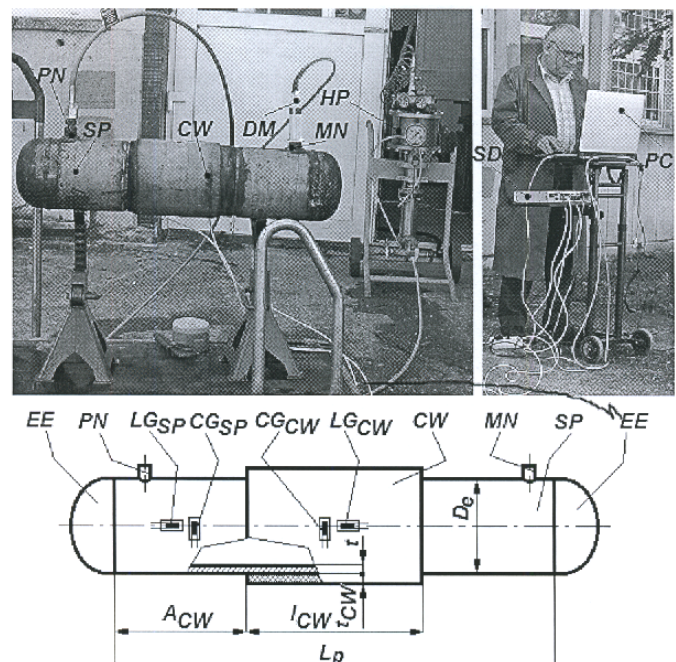


Fig. 6. Layout of TPT, prepared for the internal pressure testing



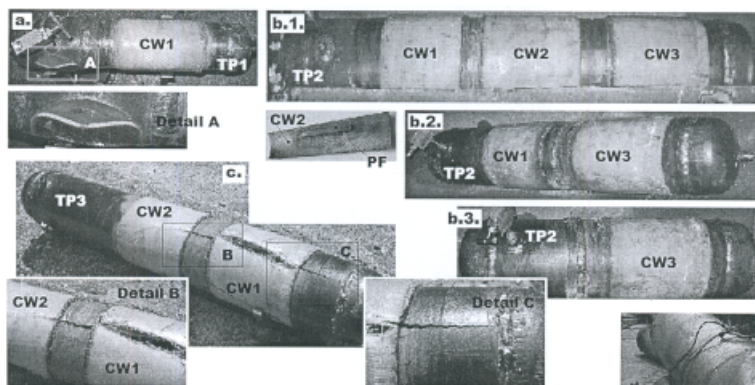
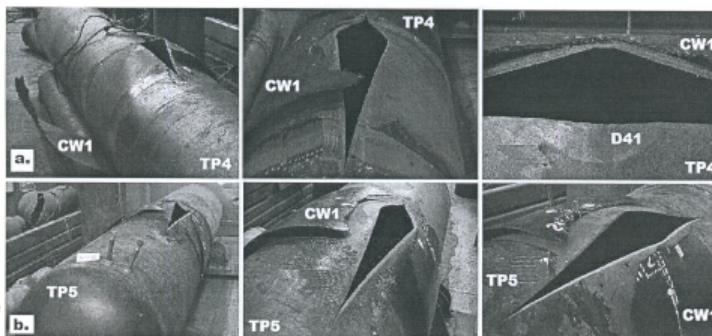


Fig. 7. TP1...TP3 layouts after the internal pressure testing up to bursting

Fig. 8. TP4 and TP5 layouts after the internal pressure testing up to bursting



– PC, provided with the software Catman<sup>®</sup> Express V3.0 (fig. 6).

### Results and discussions

The results of the experimental program regarding the TP internal pressure testing up to bursting are summarised in table 3, and the TP layout after failure is shown in figures 7 and 8. Table 3 also contains the assessments made, after performing the tests, in order to obtain the landmarks needed for the interpretation of the experimental results, regarding the values ranges of the TP burst pressure: a)  $p_{bw}$ , corresponding to the TPs with SP without defects; b)  $p_{bd}$ , corresponding to the TPs with defects executed by machining on the SP; c)  $p_{br}$ , corresponding to the TPs reinforced by applying CWs in the areas with defects. In order to obtain the ranges for  $p_{bw}$ , the burst pressures of the SPs (from which each TP has been made) have been

calculated with the 12 formulae presented in [7], the obtained values have been ordered in an increasing series and the extreme values (minimum and maximum) have been eliminated; to obtain the ranges for  $p_{bd}$  it has been proceeded similarly, only that the processed values have been the burst pressures of the SPs containing the defects made by machining, calculated applying the 11 formulae from [7], and to obtain the ranges for  $p_{br}$ , the following formula [2, 8] has been used:

$$p_{br} = \frac{2}{D_e} [R_{mp} t_{mm} + R_{mCW} t_{CW}] \quad (1)$$

considering the values from tables 1 and 2 for  $D_e$ ,  $t_{mm}$ ,  $t_{CW}$ ,  $R_m$  and  $R_{mCW}$ .

With the estimated values of the pressures  $p_{bw}$ ,  $p_{bd}$  and  $p_{br}$  and with the values determined experimentally for the TP

TP	Test sequence	Estimated values of burst pressure (MPa) for TP:			$I_{50}$ ; $I_{Rt}$ <sup>a)</sup>
		without defect $p_{bw}$	with defect $p_{bd}$	repaired with CW $p_{br}$	
TP1	Fig. 7.a	39.3 ... 41.0	17.9 ... 21.8	37.8 ... 42.4	$I_{50} = 0.445 \dots 0.563$ ; $I_{Rt} = 0.390 \dots 0.623$
TP2	Fig. 7.b.1	42.3 ... 43.3	11.9 ... 18.4	23.6 ... 27.0	$I_{50} = 0.565 \dots 0.725$ ; $I_{Rt} = 0.120 \dots 0.357$
	Fig. 7.b.2		8.7 ... 16.1	25.7 ... 29.1	$I_{50} = 0.619 \dots 0.799$ ; $I_{Rt} = 0.222 \dots 0.482$
	Fig. 7.b.3		12.7 ... 19.0	26.2 ... 29.6	$I_{50} = 0.551 \dots 0.707$ ; $I_{Rt} = 0.166 \dots 0.400$
TP3	Fig. 7.c	45.7 ... 48.5	16.1 ... 23.4	28.3 ... 31.3	$I_{50} = 0.488 \dots 0.668$ ; $I_{Rt} = 0.101 \dots 0.359$
TP4	Fig. 8.a	18.6 ... 19.3	10.1 ... 13.4	9.3 ... 9.4	$I_{50} = 0.279 \dots 0.477$ ; $I_{Rt} < 0$
TP5	Fig. 8.b	17.5 ... 18.1	10.3 ... 12.8	8.3 ... 8.4	$I_{50} = 0.293 \dots 0.431$ ; $I_{Rt} < 0$
TP	Test sequence	Burst pressure $p_b$ , MPa	Burst location	CW behaviour	$I_r$ , RE <sup>b)</sup>
TP1	Fig. 7.a	39.5	SP, near CW1	CW unbroken	$I_r = 0.432 \dots 0.550$ RE – Major
TP2	Fig. 7.b.1	29.5	SP – D22, under CW2	CW unbroken Water leaking under CW2	$I_r = 0.256 \dots 0.416$ RE – Moderate
	Fig. 7.b.2	30.0	SP – D21, under CW1	CW unbroken Water leaking under CW1	$I_r = 0.321 \dots 0.504$ RE – Moderate
	Fig. 7.b.3	31.5	SP – D23, under CW3	CW unbroken Water leaking under CW3	$I_r = 0.289 \dots 0.444$ RE – Moderate
TP3	Fig. 7.c	31.5	SP – D31 + CW1 & propagation of burst at CW2	CW1 broken & cracks on CW2	$I_r = 0.167 \dots 0.337$ RE – Minor
TP4	Fig. 8.a	14.2	SP – D41 + CW1	SP – D41 + CW1 broken	$I_r = 0.041 \dots 0.234$ RE – Insignificant
TP5	Fig. 8.b	14.9	SP – D5 + CW1	SP – D51 + CW1 broken	$I_r = 0.116 \dots 0.262$ RE – Insignificant

a)  $I_{50} = 1 - p_{bd}/p_{bw}$ ;  $I_{Rt} = (p_{br}/p_{bd} - 1)(p_{bd}/p_{bw}) = (p_{br} - p_{bd})/p_{bw}$ ; b)  $I_r = (p_b - p_{bd})/p_{bw}$ ; RE – CW reinforcement effect

**Table 3**  
RESULTS OF INTERNAL PRESSURE TESTING OF THE TPs UP TO BURSTING

burst pressures  $p_b$ , the typical ranges have been calculated (table 3) for the values of the following indices for the syntetical characterization of a TP: a)  $I_{so}$  – the softening index of the TP mechanical strength due to the presence of the defects existing on the SP; b)  $I_r$  and  $I_e$  – the index, assessed / theoretical and effective, characterizing the TP reinforcement effect, produced by applying CWs in the SP areas with defects. The reinforcement effect RE obtained by applying CWs on the TP areas with defects has been qualitatively evaluated (major, moderate, minor, insignificant) taking into account the measure in which the criteria have been fulfilled: a)  $p_b > p_{bw}$ ,  $p_{bw} > p_{bd}$ ; b)  $I_{so} \leq I_r \leq I_e$ ; in order to perform these assessments, a diagram summarising the results of the experimental program, shown in figure 9, has been built.

Regarding the particularities of completing the experimental program regarding the TP internal pressure testing up to bursting and the interpretation of the results of

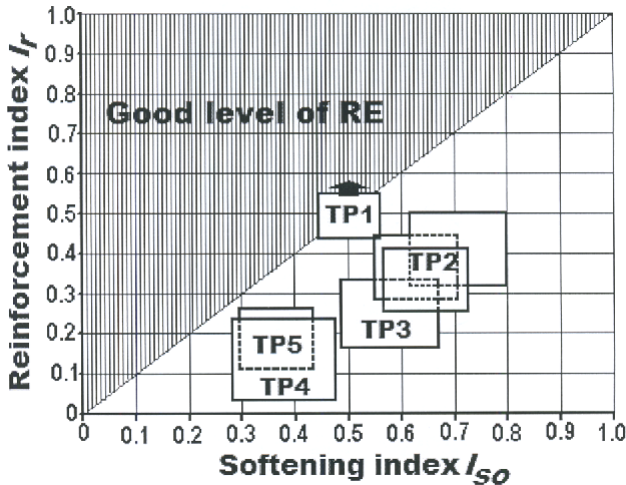


Fig.9. Results summary for the TPs internal pressure testing up to bursting and assessment of the RE of the CWs

this program, the following comments must be made: a) testing of TP1 have led to its bursting outside the area where CW has been applied (fig. 7.a) and has disclosed that  $p_b > p_{bw} \gg p_{bd}$  and  $I_{so} \approx I_r \approx I_e$ , justifying the assessment of a major RE; b) the first testing sequence of TP2 have led to its failure and tightness loss in the area of defect D22 (under CW2, fig. 7.b.1); the area in which failure occurred has been removed by cutting, TP2 has been reconstituted, joining by welding the undamaged parts (fig. 7.b.2), and its second testing sequence has been executed, leading to its failure and tightness loss in the area of defect D21 (under CW1); the area in which the second failure took place has been removed by cutting, TP2 has been reconstituted again from the undamaged parts (fig. 7.b.3) and the third testing sequence has been performed, which has led to TP2 failure and tightness loss in the area of defect D23 (under CW3); as for TP2 it has been noted that  $p_b \geq p_{br} > p_{bd}$ , but  $p_b < p_{bw}$ ,  $I_r \leq I_e$ , but  $I_{so} > I_r$  and, in addition, all CWs have not been adherent to the SP (due to the fact that the SP was unprepared by sandblasting before applying the CW), it has been estimated that the CW used to repair the TP2 defects had a moderate RE; c) testing of TP3 have led to its bursting in the area of defect D31, full breakage of CW1 applied over the defect D31, fracture propagation towards the area with defect D32 and cracking of CW2 applied over it (fig. 7.c); because TP3 testing showed that  $p_b \approx p_{br} > p_{bd}$ , but  $p_b < p_{bw}$  and  $I_{so} \gg I_r \approx I_e$ , it has been assessed that the RE of the CW applied on TP3 has been minor; d) testing of TP4 have led to its bursting in the area of defect D41, full rupture and detachment of the CW from the SP; as it has been noted that  $p_b < p_{bw}$ ,  $p_{br} < p_{bd} \approx p_b$  and  $I_{so} > I_r$ , it has been estimated that the RE of applying the CW from the composite material KPB (with a low mechanical strength, due to the weak properties of the polymeric matrix and the inadequate structure of the

fibreglass fabric, which did not assure an adequate density of the fibres in the CW) has been insignificant; e) testing of TP5 have led to conclusions similar to the ones resulting in the case of TP4.

The experimental program whose results have been described has emphasized the possibility for CWs applied in the HTP areas with defect to increase their mechanical resistance, expressed using the level of the internal pressure at which the ultimate limit state corresponding to bursting / failure of HTP pipes is reached, the conditions that must be fulfilled to that purpose being: a) the adequate selection of the composite material from which CWs are made; b) the application of CWs with the thickness  $t_{CW}$  appropriately chosen. Hence, if the CWs (made of composite material of IWR type) applied upon TP2 would have had  $t_{CW} \geq 12$  mm, and the ones applied upon TP3 would have had  $t_{CW} \geq 8$  mm, the internal pressure testing would have disclosed a similar behaviour to the one observed for TP1 (bursting / failure outside the areas reinforced with CW); however, in the case of CWs applied on TP4 and TP5, the CW thicknesses which would have led to the fulfilment of the conditions  $p_b > p_{bw} > p_{bd}$  and  $I_{so} \leq I_r$  would have been, due to the low mechanical strength of the KPB composite,  $t_{CW} > 50...60$  mm, an inapplicable solution due to the high costs and to the difficulties of obtaining a good quality (without defects, of the type gas inclusions in the polymeric matrix, which drastically diminishes the mechanical properties of the composite [2, 9]) of CWs with such great thickness.

The results of the experimental program aiming to emphasize the states of strains and mechanical stresses generated by the internal pressure loading (in the elastic region) of the TPT are summarised in figure 10. The strains  $\epsilon_{qSP}$  and  $\epsilon_{SP}$  have been obtained by recording the signals received (during the TPT loading at different levels of the internal pressure  $p$ ) from the gauges  $CG_{SP}$  and  $LG_{SP}$  and the strains  $\epsilon_{qCW}$  and  $\epsilon_{ICW}$  have been obtained by recording the signals received from  $CG_{CW}$  and  $LG_{CW}$ ; the mechanical stresses  $\sigma_{qSP}$ ,  $\sigma_{ISp}$ ,  $\sigma_{qCW}$  and  $\sigma_{ICW}$  have been calculated with the formulae:

$$\sigma_{qSP} = \frac{1}{1 - \mu_{SP}^2} (\epsilon_{qSP} + \mu_{SP} \epsilon_{ISp}) E_{SP};$$

$$\sigma_{ISp} = \frac{1}{1 - \mu_{SP}^2} (\epsilon_{ISp} + \mu_{SP} \epsilon_{qSP}) E_{SP}; \quad (2)$$

$$\sigma_{qCW} = \frac{1}{1 - \mu_{CW}^2} (\epsilon_{qCW} + \mu_{CW} \epsilon_{ICW}) E_{CW};$$

$$\sigma_{ICW} = \frac{1}{1 - \mu_{CW}^2} (\epsilon_{ICW} + \mu_{CW} \epsilon_{qCW}) E_{CW}; \quad (3)$$

considering  $\mu_{SP} = 0.3$ ,  $E_{SP} = 205$  GPa (for SP) and the values given in table 4 for the IWR  $\mu_{CW}$  and  $E_{CW}$ . In order to validate the experimental results, the diagrams from figure 10 also show the variations, as a function of  $p$ , of the stresses and strains calculated with the formulae developed in the thick-wall theory of tubes [10]: a) with continuous lines,  $\epsilon_{qSP}$  and  $\epsilon_{ISp}$ ,  $\sigma_{qSP}$  and  $\sigma_{ISp}$  on the external surface of TPT without CW; b) with dotted lines,  $\epsilon_{qSP}$  and  $\epsilon_{ISp}$ ,  $\sigma_{qSP}$  and  $\sigma_{ISp}$  on SP, at the SP – CW interface, respectively  $\epsilon_{qCW}$  and  $\epsilon_{ICW}$ ,  $\sigma_{qCW}$  and  $\sigma_{ICW}$  on the CW external surface.

The results shown in figure 10 clearly highlights that the RE of CWs applied in the HTP areas with defects, expressed by the CW capacity to unload the HTP damaged areas, is low, being equal with the ratio  $\rho_{\sigma_0}$  of the intensity of the circumferential stresses generated in the CW and in the SP upon which CWs have been applied in the conditions of HTP



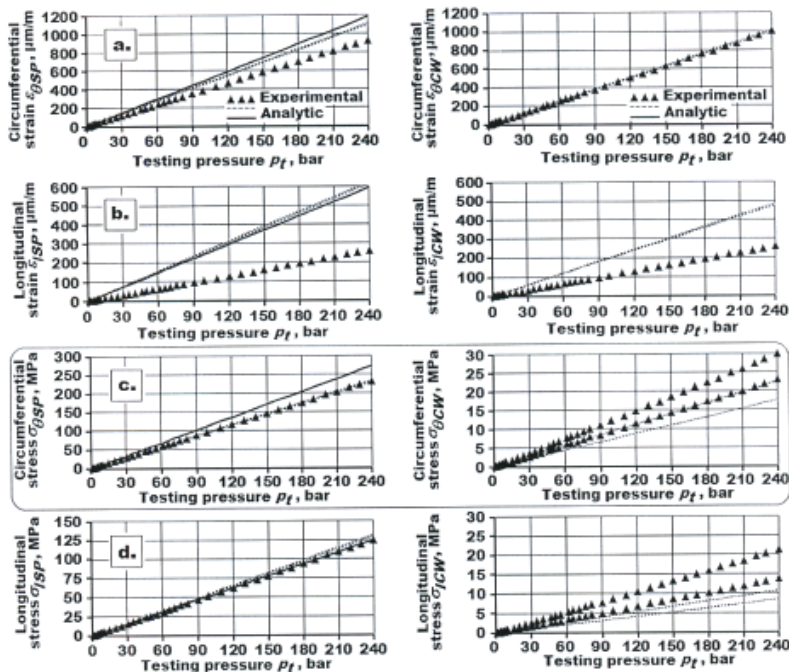


Fig. 10. The diagrams of the state of mechanical stresses and strains determined experimentally and analytically on the SP and CW of the TPT subjected to internal pressure  $p_t$  loading:  
a. circumferential strain  $\varepsilon_{\theta SP}, \varepsilon_{\theta CW} = f_1(p_t)$ ;  
b. longitudinal strain  $\varepsilon_{l SP}, \varepsilon_{l CW} = f_2(p_t)$ ; c. circumferential stress  $\sigma_{\theta SP}, \sigma_{\theta CW} = g_1(p_t)$ ; d. longitudinal stress  $\sigma_{l SP}, \sigma_{l CW} = g_2(p_t)$ ;

operating at internal pressure,  $\rho_{\sigma\theta} = \sigma_{\theta CW} / \sigma_{\theta SP} = \rho_E = E_{CW} / E_{SP}$ ; for the TPT (with CW made of IWR composite (table 1) tested within the experimental program,  $\rho_{\sigma\theta} = 0.096...0.125$  (fig. 10), and  $\rho_E = 0.085...0.110$  (and if using CWs made of last generation composites, of RES-Q Wrap type, with carbon fibres, 7...8 times more expensive than the ones with fibreglass, it is possible to ensure at the most  $\rho_{\sigma\theta} \equiv \rho_E = 0.329...0.340$ ). Remarks of the same kind, regarding the modest level of  $\rho_{\sigma\theta} = \sigma_{\theta CW} / \sigma_{\theta SP} = \rho_E$  assured by CWs made of composite materials, are also suggested in other papers [11], the increase of the CW capacity to unload the HTP damaged areas having to be one of the forefront requirements in the development of composite materials for such technical applications.

## Conclusions

The results of the experimental research programs described in this paper have led to the following conclusions:

The reinforcement effect of the composite materials wraps – CW (applied in the areas with defects of the hydrocarbon transmission pipelines – HTP), expressed using the loading conditions at which the ultimate limit state corresponding to HTP bursting / failure is reached, is dependent on the mechanical strength properties of the composite material from which CWs are manufactured and on the CWs thickness; the use of CWs made of composite material of the IWR type (or other similar materials (table 1), with thicknesses  $t_{CW} = 5...25$  mm, is suitable, as it can guarantee levels of the internal burst pressure of the HTP area repaired by applying CW superior to the burst pressure of HTP without defects, while the use of CWs made of KPB type materials is not convenient, because getting a reinforcement effect similar to the one ensured by CWs made of IWR type composites implies the usage of very thick CWs ( $t_{CW} \geq 50...60$  mm), inappropriate due to high costs and technological problems of implementation.

The reinforcement effect of the composite materials wraps – CW, expressed by the CW capability to unload (to reduce the stress intensities generated in) the HTP damaged areas, is low, being equal to the ratio  $r_E$  of the elasticity (Young's) module of the CW and of the steel from which HTP pipes are manufactured,  $\rho_E = E_{CW} / E_{SP}$ ; making the CW from composite material of the IWR type (or other

similar materials (table 1) guarantees levels of  $\rho_E = 0.085...0.110$ ; the use of CWs made of materials of the KPB type is not considered appropriate, because it does not ensure values of  $r_E$  greater than 0.015, and, if one appeals to the CW made of last generation composite materials, with  $E_{CW} = 50...70$  GPa (table 1), it is possible to reach  $\rho_E = 0.240...0.340$ .

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